

# 1 **Critical slip and time dependence in sea ice friction**

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6

## 7 **Abstract**

8 Recent research into sea ice friction has focussed on ways to provide a model which  
9 maintains much of the clarity and simplicity of Amonton's law, yet also accounts for memory  
10 effects. One promising avenue of research has been to adapt the rate- and state- dependent  
11 models which are prevalent in rock friction. In such models it is assumed that there is some  
12 fixed critical slip displacement, which is effectively a measure of the displacement over  
13 which memory effects might be considered important. Here we show experimentally that a  
14 fixed critical slip displacement is not a valid assumption in ice friction, whereas a constant  
15 critical slip time appears to hold across a range of parameters and scales. As a simple rule of  
16 thumb, memory effects persist to a significant level for 10s. We then discuss the  
17 implications of this finding for modelling sea ice friction and for our understanding of  
18 friction in general.

19 *Keywords: ice, friction, critical slip*

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## 21 **Highlights**

- 22 • Sea ice friction shows memory, which decays over a fixed time.
- 23 • A rate-and-state model can be used to quantify this memory.
- 24 • Model predictions agree well with experimentally measured dynamic friction.

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**Sea ice friction and memory effects**

The behaviour of sea ice ensembles is of scientific and engineering interest on a range of scales, from determining local forces on an ice-moored structure to predicting whole-Arctic behaviour in climate models. Sea ice deformation is controlled by friction, through ridging, rafting, and in-plane sliding. Dry friction, on the macroscopic scale, is well understood by Amonton’s law (that the ratio of shear to normal forces on a sliding interface is a constant,  $\mu$ ). Ice friction, in contrast, involves processes of melting and freezing, and associated lubrication and adhesion, and is hence somewhat more complicated. One key understanding is that when melting and freezing occur, friction can only be predicted if we know the state of the sliding interface, and hence memory effects must be included in any model.

There are two different approaches to this challenge, and progress has been made in both. The first is to work towards a better understanding of the detailed thermodynamics and micromechanics of ice friction. Work on lubrication models of ice friction has built on the foundation provided by Oksanen and Keinonen (1982); the effects of freezing have been summarised by Maeno and Arakawa (2004); the micromechanics of asperity contacts are considered by e.g. Hatton et al., 2009. The second possibility is to work on empirical adaptations of Amonton’s law to incorporate memory effects (see e.g. Lishman et al., 2009, 2011; Fortt and Schulson, 2009). It seems reasonable to believe that the two approaches are mutually compatible, and might combine to provide a clearer picture of ice friction.

One empirical adaptation of Amonton’s law which has gained significant traction in the rock mechanics literature is a rate and state friction model. Such a model accounts for two properties of friction which are frequently empirically observed:

- 1) Friction depends on the rate at which surfaces slide past each other, and
- 2) The state of the sliding surface affects the friction coefficient, and is itself affected by frictional sliding.

Friction in such models is assumed to be composed of a constant value, a rate-dependent term, and one or more state variables (see Ruina (1983) for discussion). The simplest rate and state model has the form:

$$\mu = \mu_0 + \theta + A \ln \frac{V}{V^*} \quad (1a)$$

$$\frac{d\theta}{dt} = -\frac{V}{L} \left( \theta + B \ln \frac{V}{V^*} \right) \quad (1b)$$

62 where  $\mu$  is the time-dependent effective friction coefficient,  $V$  is the slip rate,  $V^*$  is a  
 63 characteristic slip rate, and  $\theta$  is the state variable, which affects the overall friction  
 64 coefficient (equation 1a) and varies with sliding (equation 1b).  $A$ ,  $B$ , and  $\mu_0$  are empirically  
 65 determined parameters of the model. In this work, however, we wish to focus on  $L$ , the  
 66 critical slip displacement. Ruina (1983) states that one basic feature of a system which fits a  
 67 rate and state model is that “the decay of stress value after [a] step change in slip rate has  
 68 characteristic length that [is] independent of slip rate”. Ruina notes that this feature  
 69 “appears to be common to the limited recent observations” in rock mechanics. Both  
 70 Lishman et al., 2009, and Fortt and Schulson, 2009, have gone on to make the assumption  
 71 that a critical slip displacement is also a characteristic of ice friction.

72 The critical slip displacement is best understood graphically from figure 1. The upper graph  
 73 shows an instantaneous change in slip rate across a sliding interface, while the lower part  
 74 shows the typical frictional response for such a change. Qualitatively, such a response has  
 75 been shown to occur in ice (Fortt and Schulson, 2009). Under steady sliding at initial slip rate  
 76  $V_1$ , friction is steady at some constant value  $\mu_1^{SS}$ . On acceleration, friction instantaneously  
 77 increases to some value  $\mu_{peak}$ , and then gradually decays to some new steady state value  
 78  $\mu_2^{SS}$ . The critical slip displacement,  $L$ , is defined as the distance over which friction decays  
 79 from  $\mu_{peak}$  to  $[e^{-1}(\mu_{peak} - \mu_2^{SS}) + \mu_2^{SS}]$  (hereon abbreviated to  $\mu_{cs}$ ), and is shown as such on  
 80 figure 1.

81 In this work we wish to better understand the critical slip of sea ice, and so we are  
 82 particularly interested in the scaling of the frictional decay from  $\mu_{peak}$  to  $\mu_2^{SS}$ , and this region  
 83 of interest (R.O.I.) is marked with a dot-dashed line: the R.O.I. is what will be shown in later  
 84 experimental plots. Further, since we are interested in the scaling of the decay, we  
 85 normalize for  $\mu_{peak}$  and  $\mu_2^{SS}$ . Experimental plots will therefore be shown as normalized  
 86 friction  $\mu_n$ :

$$\mu_n = \frac{\mu - \mu_2^{SS}}{\mu_{peak} - \mu_2^{SS}} \quad (2)$$

87 to allow straightforward comparison across results with varying  $\mu_{peak}$  and  $\mu_2^{SS}$ .

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### 89 **The scaling of slip in sea ice**

90 We investigate the critical slip of sea ice in a series of laboratory experiments. Sea ice is  
91 grown in the UCL Rock and Ice Physics cold room facilities using carefully insulated cylinders  
92 to ensure a vertically oriented columnar ice structure comparable to that found in nature,  
93 with typical grain dimensions 10mm in the horizontal (x-y) plane and 50mm in the vertical  
94 (z) direction (see Lishman et al., 2011 for further details and thin sections). The ice is then  
95 cut to approximate shape using a bandsaw and milled to 100 $\mu$ m precision. Figure 2 shows  
96 the experimental setup, with three ice blocks (300 $\times$ 100 $\times$ 100mm) in a double shear  
97 configuration. The sliding faces are in the x-z plane, analogous to the sliding of floating ice  
98 floes in nature. One key distinction between experiment and nature is that the experiment  
99 occurs out of the saline water, and so to minimise brine drainage we conduct all  
100 experiments within 4 hours of removing the ice from water. Table 1 gives further details of  
101 the ice properties. Normal load is provided by a hydraulic load frame, while shear load is  
102 provided by a hydraulic actuator. The entire experiment occurs within an environmental  
103 chamber in which temperature can be controlled. All loads and displacements are  
104 monitored at sub-100ms intervals using externally calibrated load cells and displacement  
105 transducers.

106 Twelve experiments were run with this experimental setup and various environmental  
107 conditions, and the relevant conditions for each experiment are given in table 2. The same  
108 ice blocks were used throughout. In each experiment the central block is moved 30mm,  
109 under normal load, to ensure a repeatable sliding surface. Motion is then stopped for a  
110 given hold time (listed for each experiment in table 2): this gives  $V_1=0$ . Motion is then  
111 instantaneously resumed at some slip rate  $V_2$ , again given for each experiment in table 2.  
112 This leads to a frictional decay profile similar to that shown in figure 1. Figure 3a shows a  
113 typical actuator velocity profile for an experiment with  $V_2=1\text{mms}^{-1}$ , and we note that the  
114 laboratory actuator acceleration is around  $1\text{mms}^{-2}$ . Normalised frictional decays are shown  
115 for all experiments with  $V_2=0.1\text{mms}^{-1}$  in figure 3b, and for all experiments with  $V_2=1\text{mms}^{-1}$  in  
116 figure 3c. For each experiment  $\mu_{\text{peak}}$  and  $\mu_2^{\text{ss}}$  are given in table 2 so that normalised friction  
117  $\mu_n$  can be reconverted into absolute friction. The contrast between figure 3b and figure 3c is  
118 clear. Although the critical slip in figure 3b is somewhat obscured by secondary stick-slip  
119 behaviour (cf. Fortt and Schulson, 2009), the decay from peak friction (1 on the normalised

120 scale) to steady state friction clearly occurs within the first 1mm of slip. In contrast the  
121 equivalent decay in figure 3b occurs over around 10mm of slip. This holds true independent  
122 of hold time, temperature or side load.

123 However, it seems plausible that this difference in critical slip displacement is related to the  
124 stick-slip behaviour which occurs at low speeds. To test this hypothesis we compare our  
125 results from the UCL experiments to a series of ice tank experiments undertaken at the  
126 HSVA facility in Hamburg, Germany in the summer of 2008. In these experiments the sliding  
127 interfaces are 2m long, and the slip rate is  $16\text{mms}^{-1}$ . The normal load is provided by  
128 pneumatic load frames and the shear load by a mechanical pusher carriage. Full  
129 experimental details can be found in Lishman et al, 2009. Results from these experiments,  
130 directly comparable to those of experiments 1-12, are shown in figure 3d. Here we see that  
131 at the higher slip rate the critical slip displacement increases to roughly 120mm.

132 The results from these experiments, across different scales, strongly suggest that the critical  
133 slip displacement of ice is not a constant. Moreover, the apparently linear increase of critical  
134 slip displacement with slip rate suggests that there may be a relevant critical slip time which  
135 governs all the observed slip decays. A simple exponential decay with time is overlaid on  
136 each of the plots:

$$\mu_n = e^{-0.32t} \quad (3)$$

137 and this decaying exponential is a good representation of the frictional decay in each case.

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### 139 **Relevance to modelling friction**

140 The results of this experimental study suggest that a critical slip displacement is not a valid  
141 assumption for sea ice. It is therefore unlikely that the same rate and state models used for  
142 rock friction will be useful for sea ice friction. However, the principles behind such a model  
143 still apply: log-linear rate dependence of friction has been shown to be a useful  
144 simplification (Lishman et al., 2009; Fortt and Schulson, 2009), and memory effects have  
145 been shown to be important (Lishman et al., 2011, as well as the current work). It therefore  
146 seems worth pursuing a new model of state dependence which allows for a critical slip time  
147 rather than a critical slip displacement. One simple way to do this is to replace the  $(-V/L)$

148 term in equation 1b with a term  $(-1/t_c)$ , which maintains dimensional consistency. Doing  
 149 this, we get a new rate and state law:

$$\mu = \mu_0 + \theta + A \ln \frac{V}{V^*} \quad (4a)$$

$$\frac{d\theta}{dt} = -\frac{1}{t_c} \left( \theta + B \ln \frac{V}{V^*} \right) \quad (4b)$$

150 We can then test this new law against both the previous, displacement-focussed rate and  
 151 state law, and experimental results for friction under dynamic sliding conditions. Lishman et  
 152 al., 2011, present data from such a dynamic sliding experiment conducted in the laboratory  
 153 at  $-10^\circ\text{C}$  using the experimental configuration of figure 2 and the slip rate profile shown in  
 154 figure 4a. Here we repeat this experimental data in figure 4b, showing alongside it the  
 155 predictions of both the standard rate and state model (equation 1) and the new critical time  
 156 dependent rate and state model (equation 3). In both cases  $\mu_0=0.872$ , and the rate-  
 157 dependence term  $B-A = 0.072$  (see Lishman et al., 2011, for the origin of these parameters).  
 158  $V^*$  is a characteristic velocity for dimensional consistency: we use  $V^* = 10^{-5}\text{ms}^{-1}$ , as in  
 159 Lishman et al., 2011. For the original model  $L=0.2\text{mm}$  (experimentally measured) and  
 160  $A=0.31$  (fitted). For the new model,  $(1/t_c)$  must match the coefficient of exponential decay of  
 161 equation 3, and so  $t_c=3\text{s}$  (to 1 significant figure, for simplicity). We find  $A = 0.05$  matches  
 162 experimental data well with the new model (this value leads to instability in the original  
 163 model). In figure 4b we see clearly that the assumption of a critical slip displacement is  
 164 flawed, and that with the assumption of a critical slip time the limited friction decay on  
 165 deceleration (at  $\sim 8\text{mm}$  on fig 4b), the two stage frictional increase during acceleration and  
 166 steady state sliding ( $\sim 8\text{-}10\text{mm}$ ), the rounded frictional peak ( $\sim 10\text{mm}$ ) and the long ( $\sim 10\text{s}$ )  
 167 frictional decay under steady state sliding ( $\sim 10\text{-}20\text{mm}$ ) are all best modelled by the new rate  
 168 and state equations. We therefore conclude that sea ice friction is best modelled as having a  
 169 critical slip time, and that the standard rate and state equations, adapted to reflect this,  
 170 accurately model dynamic sea ice friction.

171 We also note two important caveats. Firstly, the memory effects encapsulated by equation  
 172 3 are necessarily restricted to incorporate the events of the previous 10s or so. For dynamic  
 173 sliding in the various scales investigated here, this seems to be a useful model. However, we  
 174 know that at zero slip rate (and by continuity at very low slip rates) consolidation occurs,  
 175 and that this process has a memory much greater than 10s (i.e. events over 10s in the past

176 can still affect the present). A complete model of sea ice friction would therefore require a  
177 second state variable, which would account for these low-slip-rate friction healing effects.  
178 This model would also make some intuitive sense, with one catch-all state term covering  
179 lubrication effects at non-zero slip rates, and another state term covering consolidation  
180 effects at slip rates very close to zero.

181 Secondly, we note from Fortt and Schulson, 2009, that the assumption of velocity-  
182 weakening (that is, decreasing friction with increasing slip rate) is only valid for slip rates  
183 above about  $10^{-5}\text{ms}^{-1}$ , and below this value our proposed model is no longer valid.

184 A further caveat is that the parameterisation used in this study will be dependent on  
185 environmental conditions. In particular, we believe that temperature will affect frictional  
186 memory, although that hypothesis is not supported by this study (perhaps because our  
187 temperature range is small compared to the absolute melting point of ice). One intriguing  
188 possibility is that the findings of this study may be relevant to crystalline materials other  
189 than ice, provided those materials are at a homologous temperature (in this case  $T \approx 0.96$   
190  $T_m$ ). Rice (2006) observes that earthquake dynamics are controlled by extremely narrow  
191 shear zones, in which significant thermal weakening occurs and the rock may indeed be at a  
192 homologous temperature to the sea ice studied in the present work. It is somewhat difficult  
193 to run laboratory rock friction experiments at temperatures close to melting: however, it is  
194 much easier to run laboratory ice friction experiments at very low temperatures well away  
195 from the melting point ( $T \approx 0.8 T_m$ , or around  $-50^\circ\text{C}$ ) and this seems a promising route for  
196 further research along the lines of the present study.

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## 198 **Conclusions**

199 The critical slip of sea ice (at temperatures close to melting) has been assumed to be over a  
200 fixed displacement but actually occurs over a fixed time. The experiments outlined in this  
201 study have shown that this critical slip time remains constant over a range of slip rates. A  
202 simple rule of thumb for engineering purposes is that memory effects in ice friction decay by  
203 a factor of  $1/e$  over 3s, and are negligible beyond 10s. This understanding can then be used  
204 to adjust a standard first order rate and state friction model, and this new model provides  
205 an excellent prediction of dynamic friction. The model has the further advantage of  
206 computational simplicity, and provides an empirical bridge between Amonton's law and  
207 more detailed physical explanations of the micromechanical controls on ice friction. A

208 second order rate and state model might also be able to incorporate the effects of healing  
209 at very low slip rates. Further work may answer the question of whether the friction  
210 behaviour described in this work is a quirk of columnar sea ice, or whether it may apply  
211 more generally to crystalline materials close to their melting temperature.

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## 215 **Acknowledgments**

216 This work was funded by the National Environmental Research Council. The ice tank work  
217 was supported by the European Community's Sixth Framework Programme through the  
218 grant to the budget of the Integrated Infrastructure Initiative HYDRALAB III, contract  
219 022441(RII3). The authors would like to thank the Hamburg Ship Model Basin (HSVA),  
220 especially the ice tank crew, for the hospitality and technical and scientific support. D.F.  
221 would like to thank the Leverhulme Trust for the award of a prize that made his  
222 participation in the HSVA experiments possible. The authors would like to thank Steve Boon,  
223 Eleanor Bailey, Adrian Turner, and Alex Wilchinsky for their contributions to the ice tank  
224 work.

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227 **References**

228 Fortt, A.L., and Schulson, E.M., 2009. Velocity-dependent friction on Coulombic shear faults  
229 in ice. *Acta Materiala*, 57, 4382-4390.

230 Hatton, D.C., Sammonds, P.R., and Feltham, D.L., 2009. Ice Internal Friction: Standard  
231 Theoretical Perspectives on Friction Codified, Adapted for the Unusual Rheology of Ice, and  
232 Unified. *Philosophical Magazine*, 89:31, 2771-2799.

233 Lishman, B., Sammonds, P.R., Feltham, D.L., and Wilchinsky, A., 2009. The Rate- and State-  
234 Dependence of Sea Ice Friction. In proceedings of Port and Ocean Engineering under Arctic  
235 Conditions, 2009.

236 Lishman, B., Sammonds, P.R., and Feltham, D.L., 2011. A rate and state friction law for saline  
237 ice, *J. Geophys. Res.*, 116, C05011, doi:10.1029/2010JC006334.

238 Maeno, N., and Arakawa, M., 2004. Adhesion Shear Theory of Ice Friction at Low Sliding  
239 Velocities, Combined with Ice Sintering. *Journal of Applied Physics*, 95, 134-139.

240 Oksanen, P., and Keinonen., J., 1982. The Mechanism of Friction of Ice. *Wear*, 78, 315-324.

241 Rice, J. R. (2006). Heating and weakening of faults during earthquake slip. *J. Geophys. Res.*,  
242 111, B05311.

243 Ruina, A. (1983). Slip instability and state variable friction laws. *J. Geophys. Res.*, 88, 10359-  
244 10370.

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<b>Location</b>	<b>Laboratory</b>	<b>Ice tank</b>
Ice thickness (m)	0.1	0.25
Water salinity (ppt)	33	33
Bulk ice salinity (ppt)	10.8	7.3
Ice density (kg m <sup>-3</sup> )	930	931

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247 Table 1: Experimental ice details

Experiment Number	Location	Temp. / °C	Slip Rate $V_2 / \text{mms}^{-1}$	Hold Time / s	Normal Load / N	$\mu_2^{SS}$	$\mu_{\text{peak}}$
1	UCL	-10	0.1	100	500	0.82	1.37
2	UCL	-10	0.1	100	1000	0.85	1.35
3	UCL	-2	0.1	100	500	0.69	1.07
4	UCL	-2	0.1	10	500	0.73	0.99
5	UCL	-10	1	100	500	0.60	1.01
6	UCL	-10	1	100	500	0.57	1.14
7	UCL	-10	1	100	1000	0.60	1.18
8	UCL	-10	1	10	500	0.87	1.10
9	UCL	-10	1	1000	500	0.59	1.28
10	UCL	-2	1	100	500	0.40	0.84
11	UCL	-2	1	10	500	0.25	0.51
12	UCL	-2	1	1000	500	0.24	0.76
13	HSVA	-10	16	100	600	0.39	0.78
14	HSVA	-10	16	100	600	0.47	1.12

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249 Table 2. Experimental configurations.

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260 **Figure Captions**

261 Figure 1. Idealised evolution of friction  $\mu$  as a function of slip displacement, for constant normal load,  
262 under an instantaneous increase in slip rate (after Ruina, 1983.) The dash-dotted box shows the  
263 region in which our later experiments are plotted.

264 Figure 2. Schematic of experimental apparatus. The ice blocks are milled to dimensions  $300 \times 100 \times$   
265  $100\text{mm}$ . The entire apparatus shown is housed in a temperature-controlled environmental chamber.  
266 The actuator is controlled hydraulically. The x-y plane facing us is the upper surface of the ice.

267 Figure 3a. Slip rate profile, as a function of time, for an experiment with  $V_1 = 0$  and  $V_2 = 1\text{mms}^{-1}$ . The  
268 solid line shows the programmed actuator speed, while the markers show the measured actuator  
269 speed. The actuator acceleration is around  $1\text{mms}^{-2}$  in the laboratory experiments.

270 Figure 3b. Time evolution of friction for experiments 1-4 (see table 2) with  $V_1 = 0$  and  $V_2 = 0.1\text{mms}^{-1}$ .

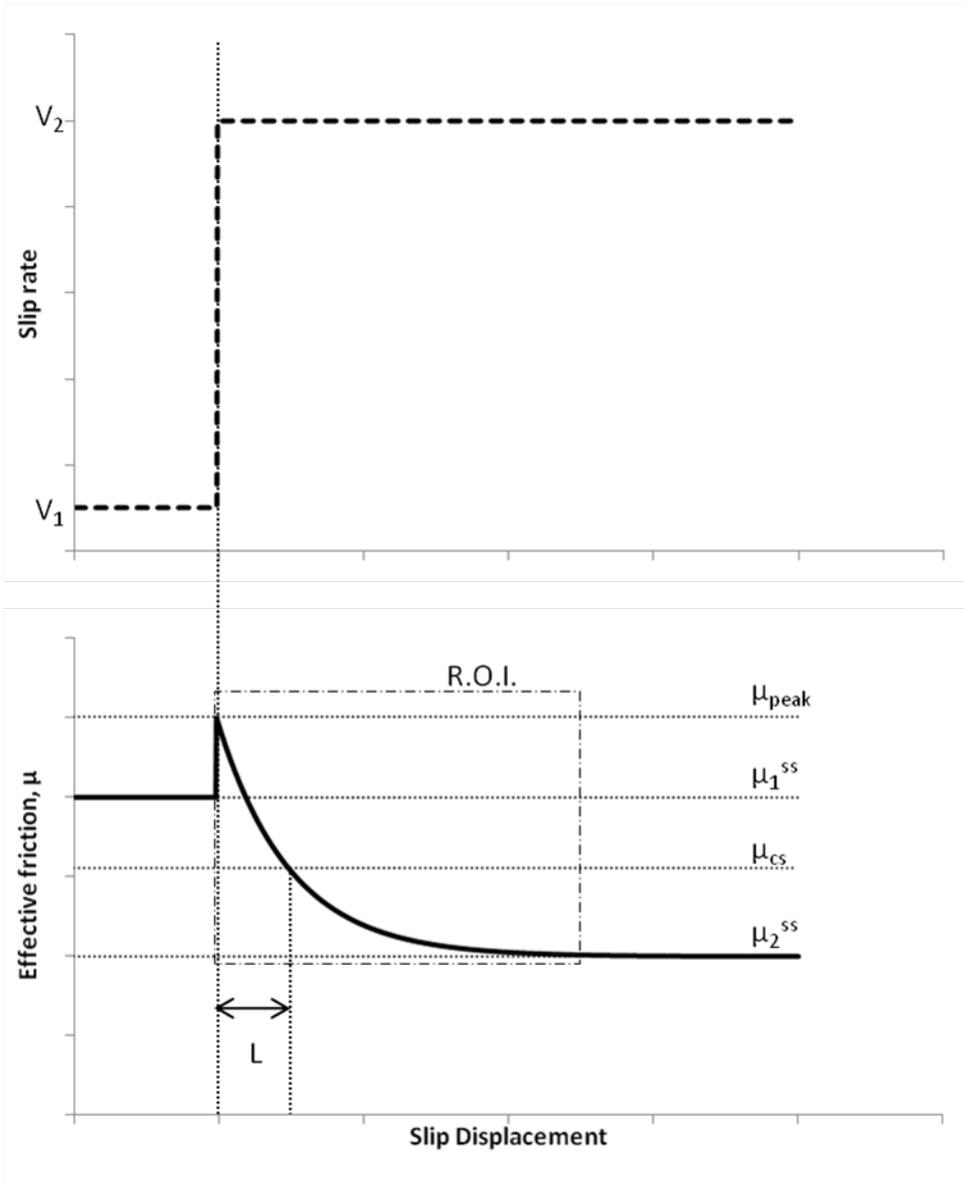
271 Figure 3c. Time evolution of friction for experiments 5-12 (see table 2) with  $V_1 = 0$  and  $V_2 = 1\text{mms}^{-1}$ .

272 Figure 3d. Time evolution of friction for experiments 13 and 14 (see table 2) with  $V_1 = 0$  and  $V_2 =$   
273  $16\text{mms}^{-1}$ .

274 Figure 4a. Slip rate profile, as a function of time, for dynamic sliding experiments. The diamond  
275 markers show the measured slip rate during the experiment, while the solid line shows the linear  
276 approximation used to model the profile.

277 Figure 4b. Comparison of the predicted friction under the standard rate and state model (grey,  
278 short-dashed line) and the new critical time dependent model (black, long dashed line) to  
279 experimental measurements. The measurements shown are from a laboratory experiment at  $-10^\circ\text{C}$ ,  
280 over the varying slip profile shown in figure 4a.

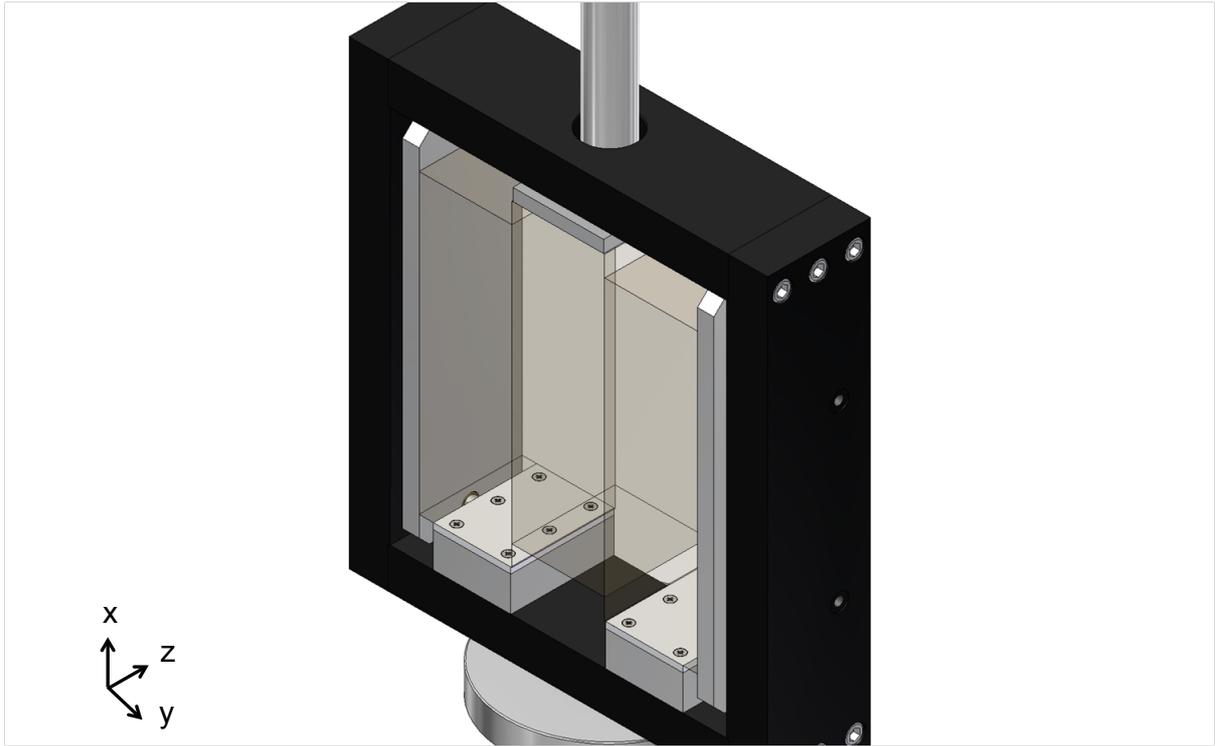
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283 FIGURE 1

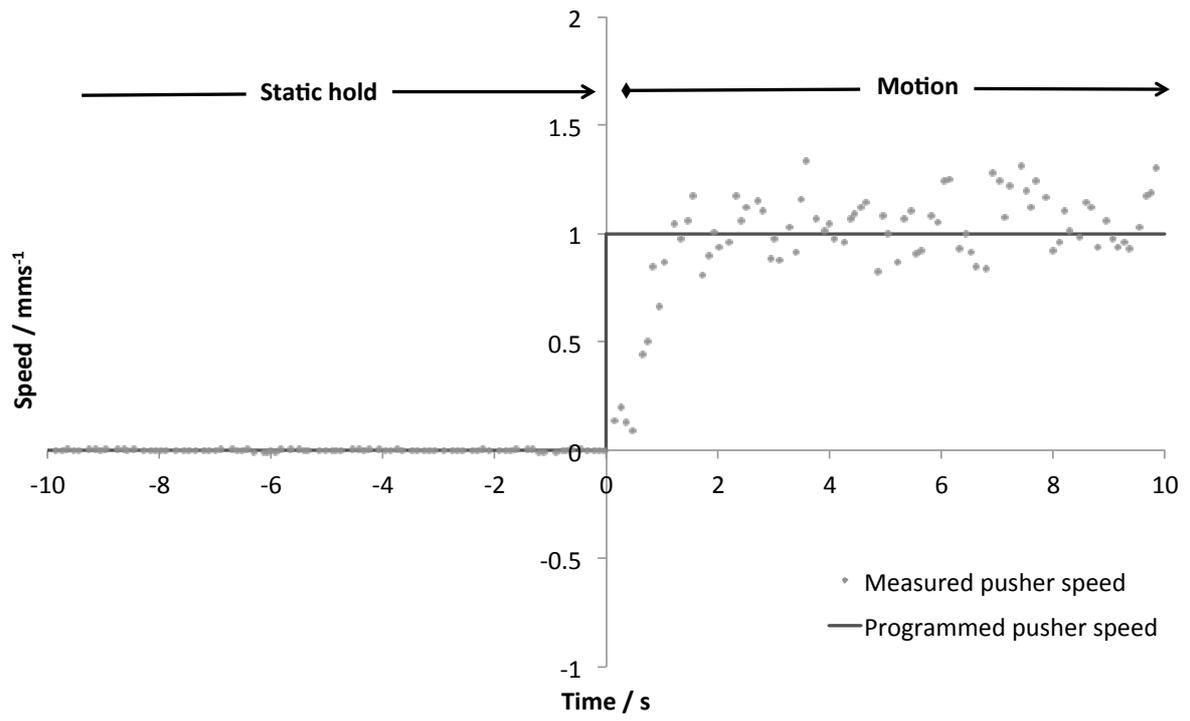
284



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286 FIGURE 2

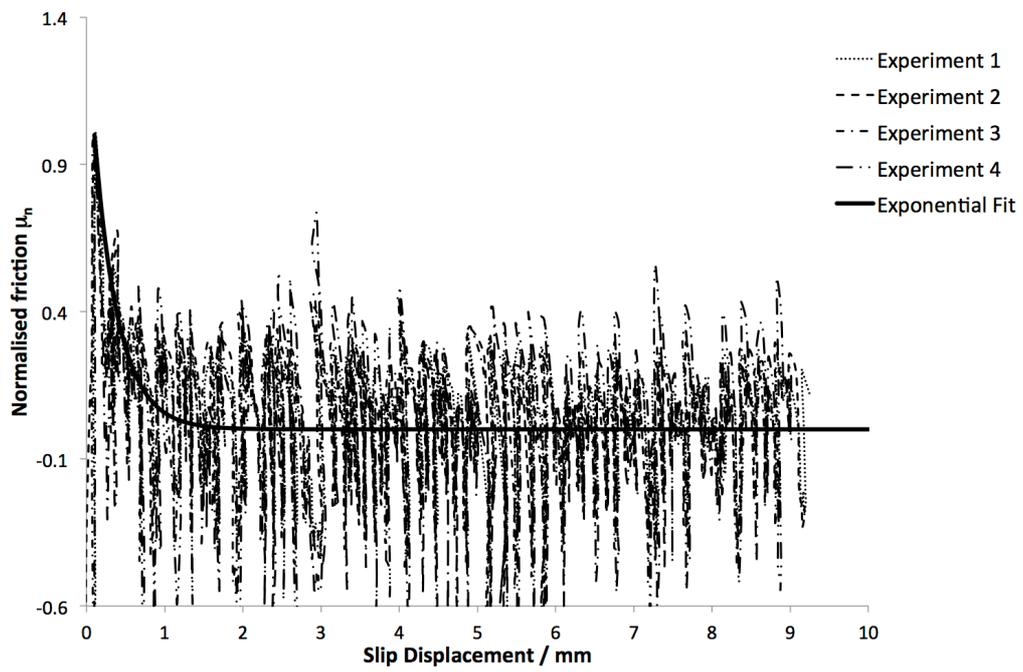
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289 FIGURE 3a

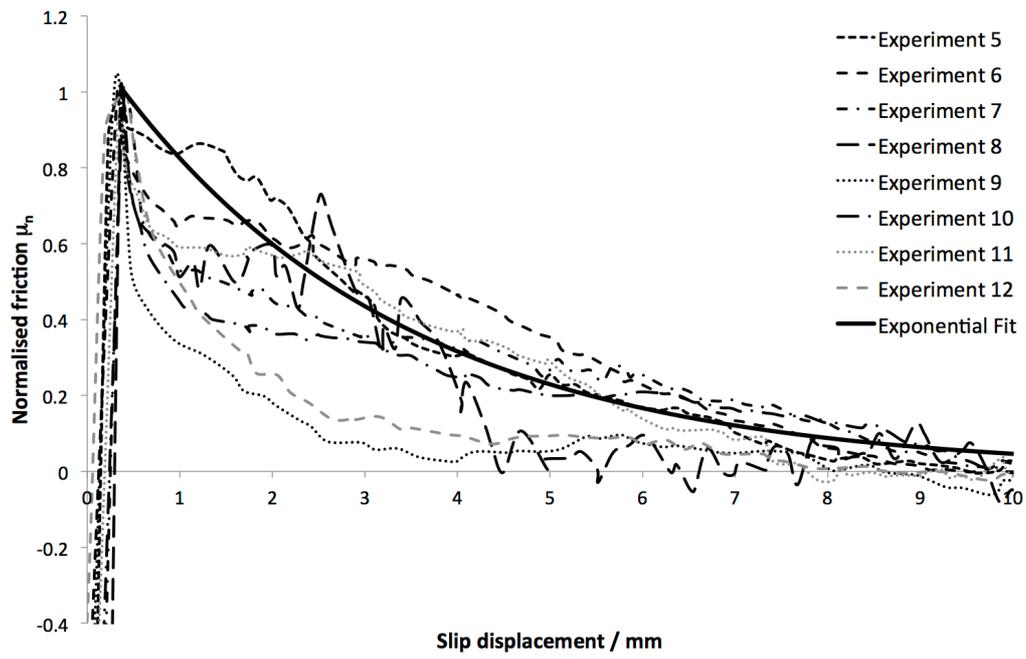
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292 FIGURE 3B

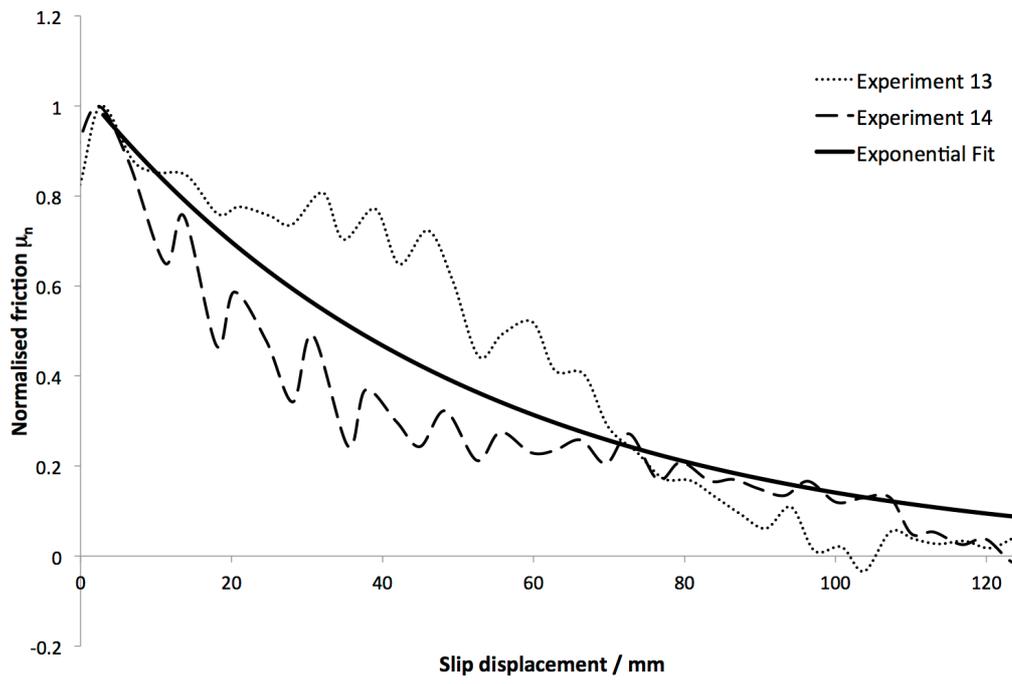
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295 FIGURE 3C

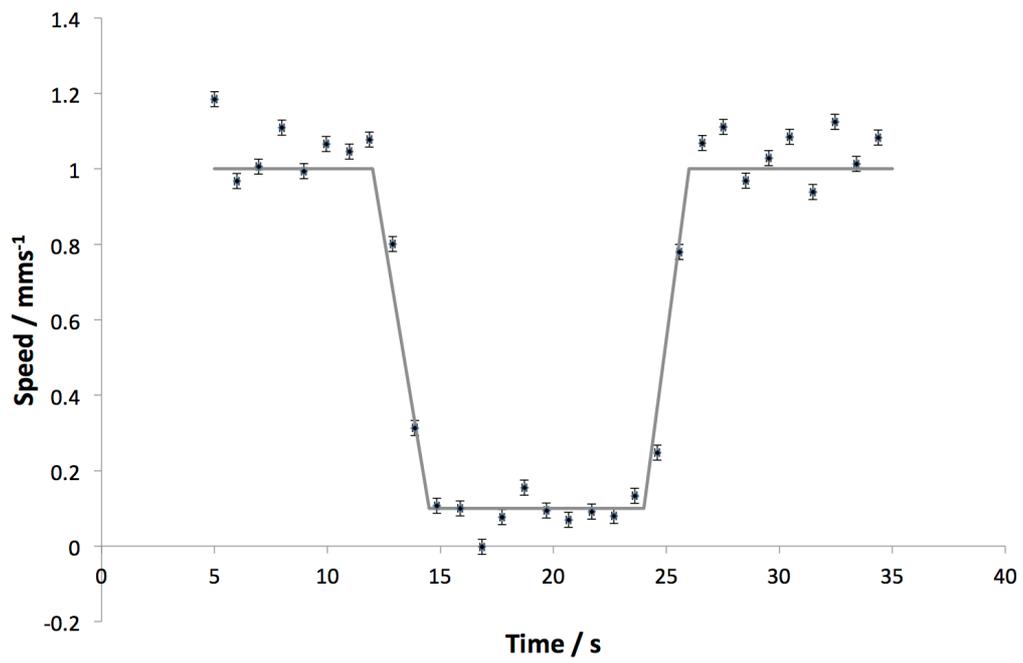
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298 FIGURE 3D

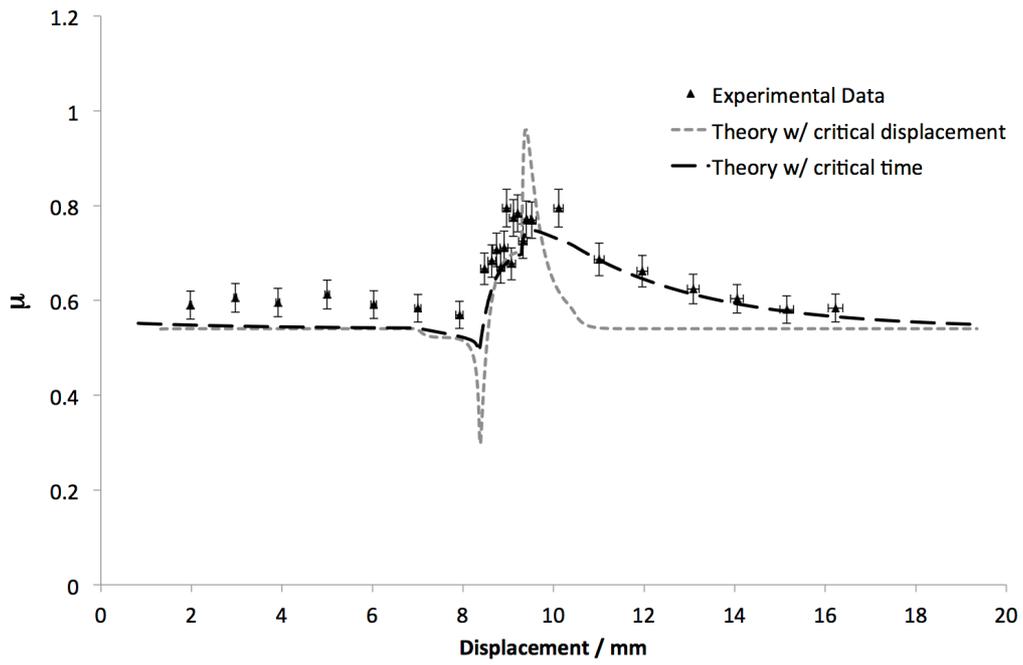
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301 FIGURE 4A

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304 FIGURE 4B

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